



# Design of a High-Speed Disc Chopper Test Rig

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# Overview

- Background
- Scope & Requirements
- Initial Concepts
- Project Plan
- Design Considerations
- Analysis
- Final Design



# The ISIS Facility

- Oxfordshire, UK
- 1985 Facility Opened
- 2008 TS2 opened
- Pulsed Spallation Source
- Target Stations
  - TS1 160kW 40Hz
  - TS2 40kW 10Hz



# The ISIS Facility

- 30 Operational Beamlines
- 350 staff members
  - 24 Instrument Design Engineers

## Types of Instrument at ISIS

- Diffractometer
- Reflectometer
- Small Angle Scattering
- Indirect Spectrometer
- Direct Spectrometer
- Muon Spectrometer/Instrument
- Chip Irradiation
- Imaging and Diffraction



# Background

- New FREIA instrument
- ESS require larger disc choppers than any at ISIS
- Currently no facility for over-speed testing of disc choppers (balancers, acoustic chambers)
- Use analysis and past experience



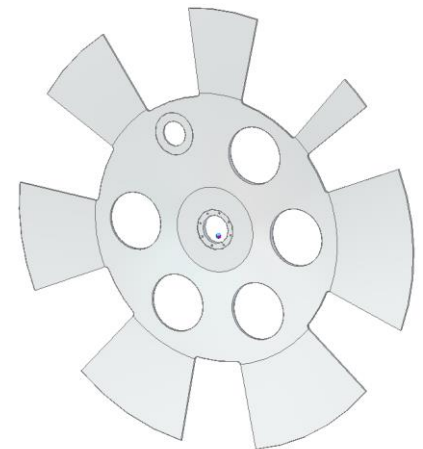
EUROPEAN  
SPALLATION  
SOURCE



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# Disc Chopper

- Disc choppers allow the flow of selected neutrons, and absorb all others.
- Aluminium alloy disc potted with an area of neutron absorbing B<sub>4</sub>C resin.
- Rotating at up to 50hz – 3000rpm.
- Synchronised to the beam pulses.





# Requirements

Requirement	Essential	Desired
Capable of testing chopper discs of varying size	Up to 1.3 metre diameter	Up to 1.5 metre diameter
Fits within space allocated for rig	AxBxC m of room	Possible to change motors/discs/fittings with ease of movement in space
Parts of rig needing moving capable of being lifted by crane	Proposed new crane limit 2 ton	-----
Mechanism for opening rig to change chopper discs and motor	-----	Can be performed with minimal crane assistance
Double disc testing	-----	
Service life	-----	10 years
Vacuum	$10^{-3}$ mbar	
Door	Contain any catastrophic failure	Double hinge or craned in. Operable by 1 person
Mechanism to put disc in place – crane	Capable of placing 1.3-1.5 meter (20-50 kg) chopper disc on chamber end easily	-----
Rig secured to floor	Bolted directly to ground to negate dynamics/vibration effects	-----
Adaptor to connect motor bearing system	SKF G5 adaptor	Other adaptors to be manufactured to accommodate a variety of motors
Modular design	Rig can be disassembled and rebuilt in new location.	-----
Ports for monitoring of discs	Power feedthroughs for thermocouples and speed monitors	Possibility of adding a viewport

# Location

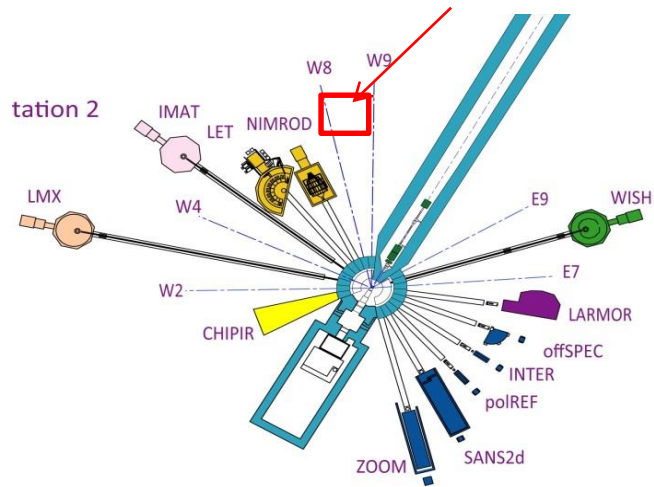


Figure 1: Proposed location of test rig.

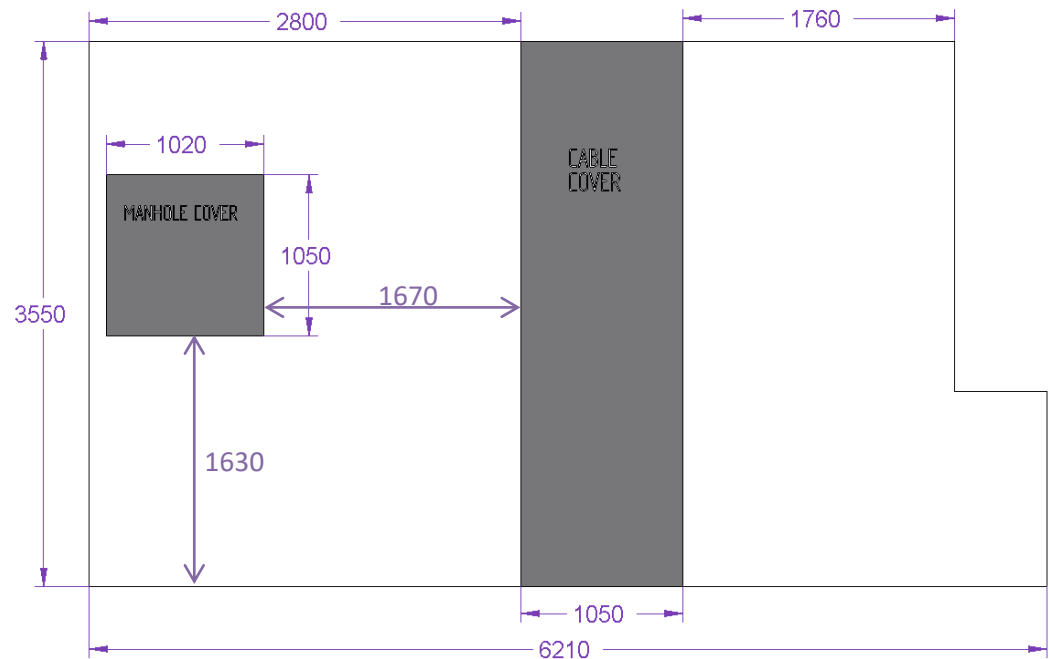
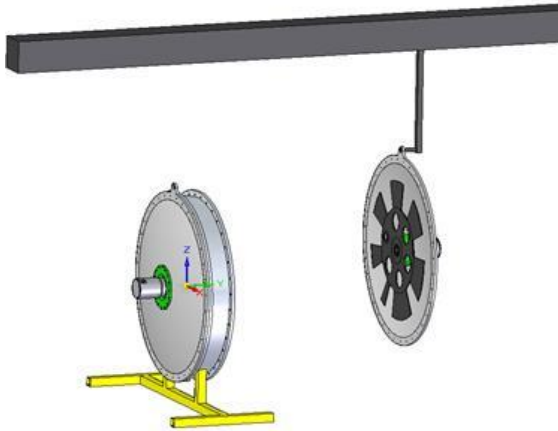


Figure 2: Floor dimensions of bunker, in mm, to house the rig (manhole cover and cable trench cover shown)

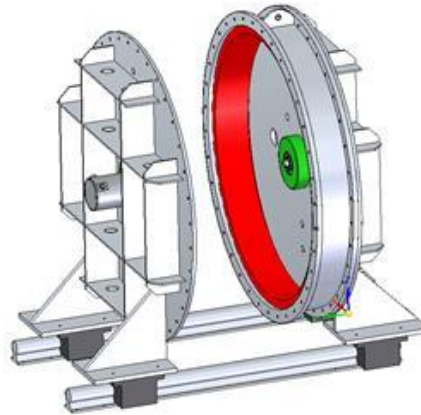




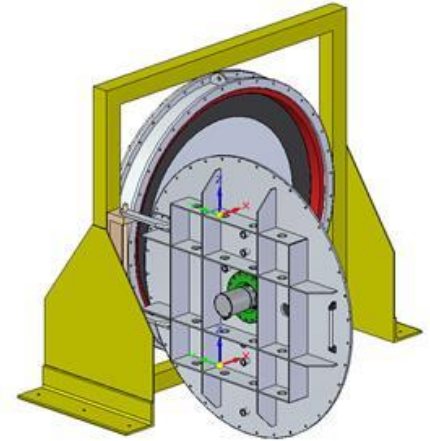
# Initial Concepts



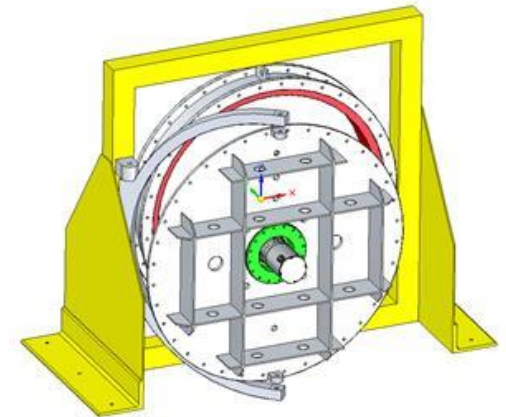
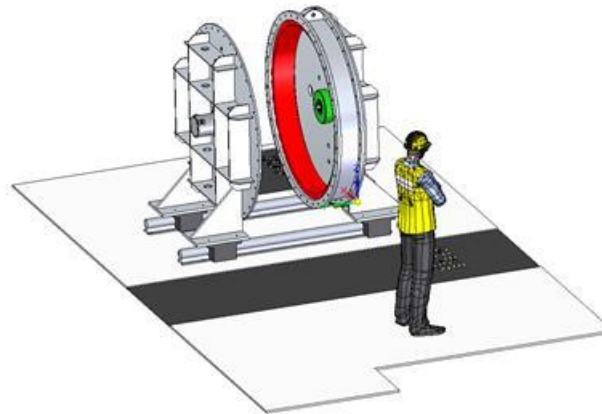
*Stand-alone concept*



*Rails concept*



*Door concept*

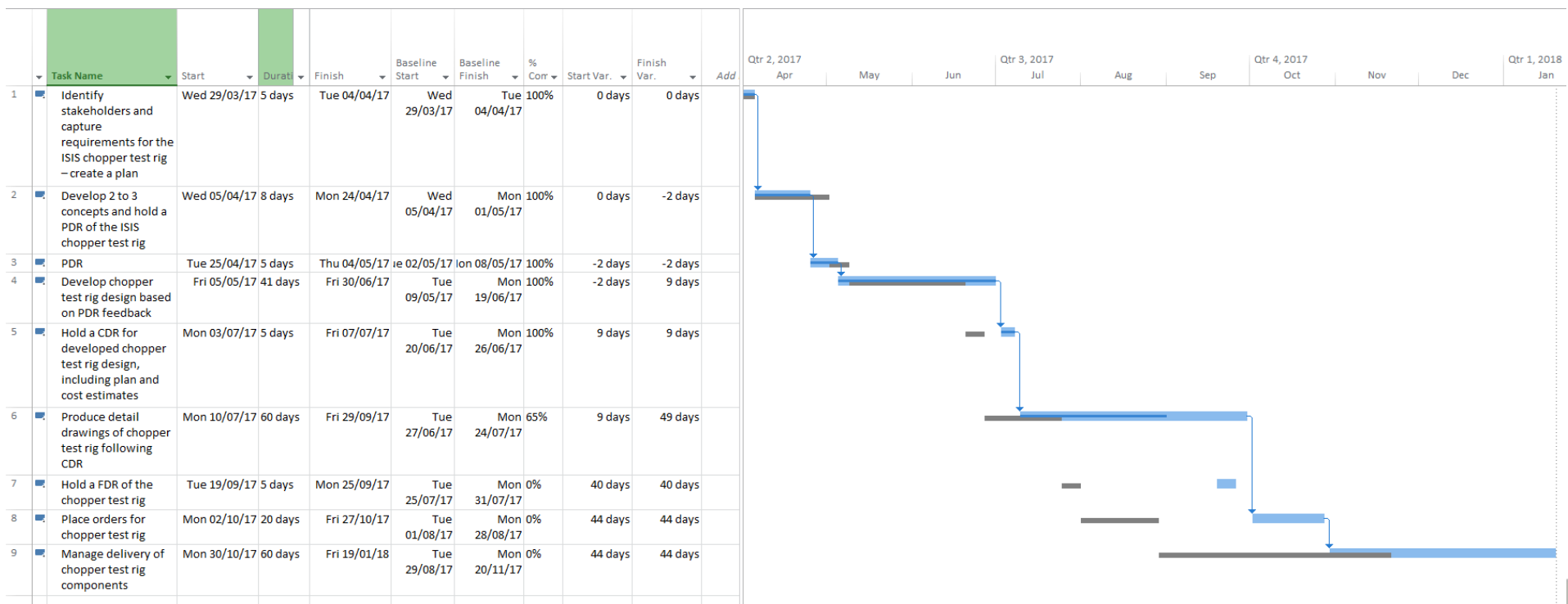




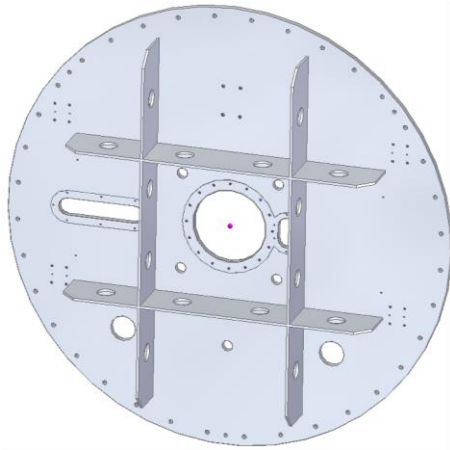
# Project Plan

- March 2017 – Scope & Stakeholders established
- Concept designs
- PDR – End of April
- Develop design based on PDR
- CDR – Start of July
- Create drawings \*Largest deviation from plan\*
- October 2017 – FDR held and Progress document written

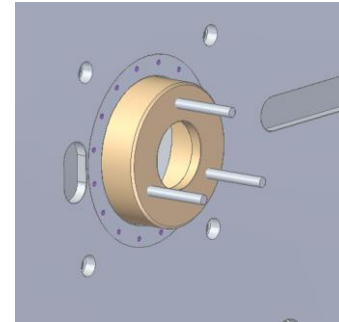




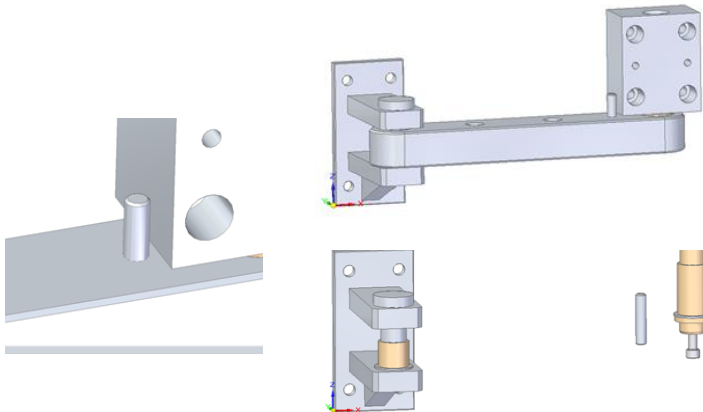
# Design Considerations



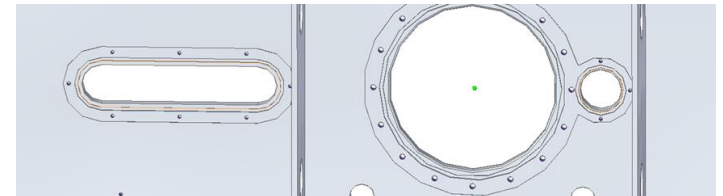
*Ribs*



*Guide Rods*



*Hinge System*

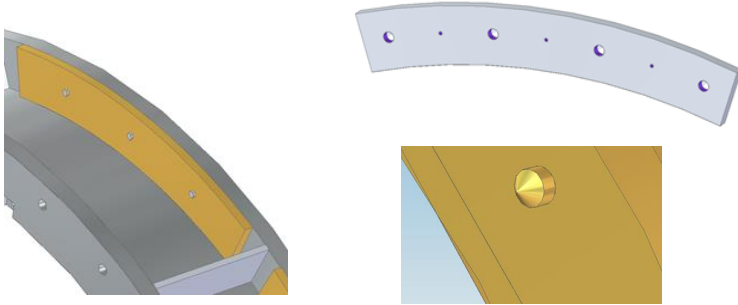


*Access Holes*

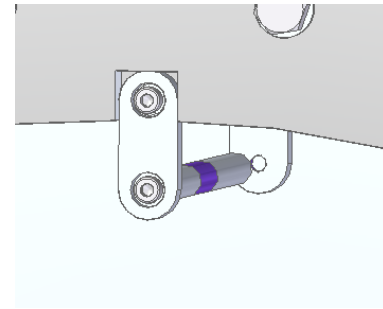




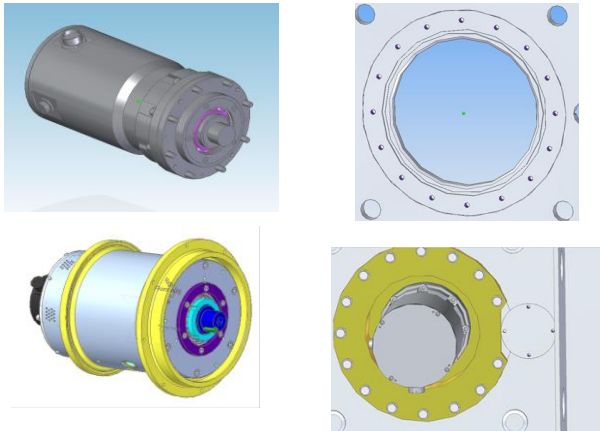
# Design Considerations



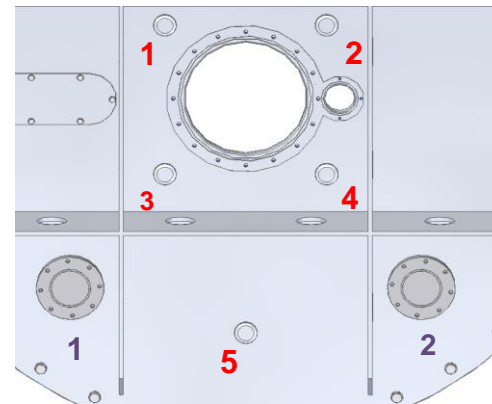
*Screw Plates*



*Baffle Plate connection*



*Adaptor Hole*



KF40

ISO-F 100

*Vacuum Ports*



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# Analysis

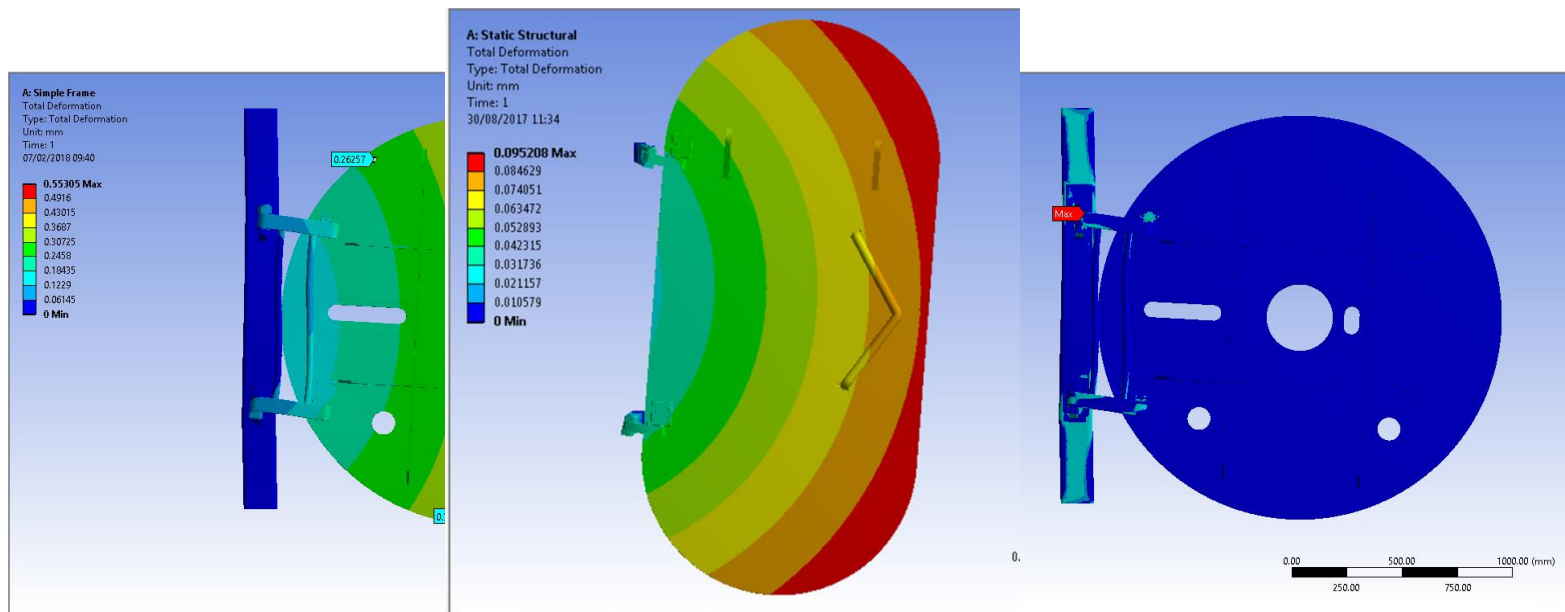
- Analysis and stress calculations were performed on aspects of the test rig
  - Deformation of door
  - Deformation of chamber
- Led to changes in design
  - Castor added to bottom of door so it can close
  - Hinge system redesigned to be more rigid





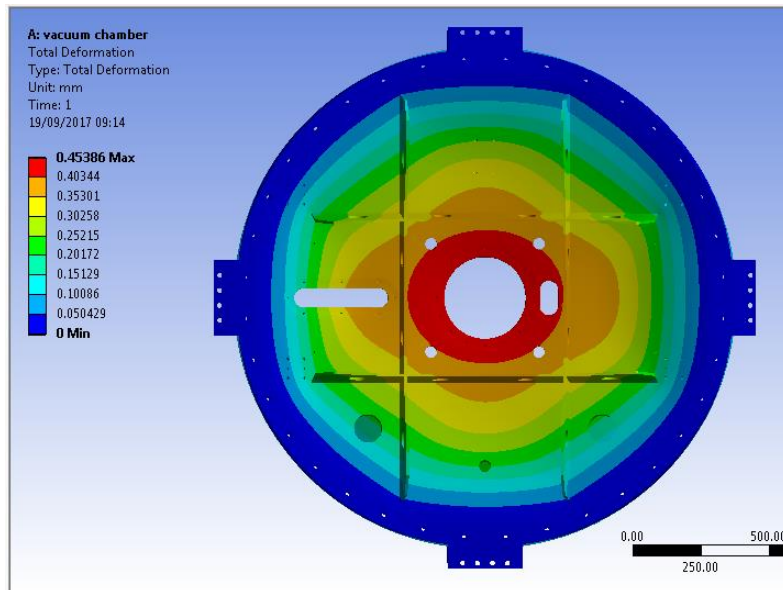
# Hinge System Analysis

- Ideally deformation less than  $\sim 0.1$  mm so door can close easily
- After analysis and changing of parameters decided to add a jacking castor
- Reduced deformation and easier operation of door

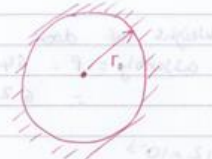


# Pressure Vessel

- Deformation of fixed plate under pressure
- Hand calculations completed
- Deformation of ~0.45 mm
- Regular shape of deformation important



Simple Clamped Circular Plate



Max deflection at centre =  $\frac{Pr_0^4}{64D}$

$P = 1 \times 10^5 \text{ Pa}$   $r_0 = 0.8125 \text{ m}$   
 $t = 30 \times 10^{-3} \text{ m}$   $E = 200 \text{ GPa}$   
 $\nu = 0.3$

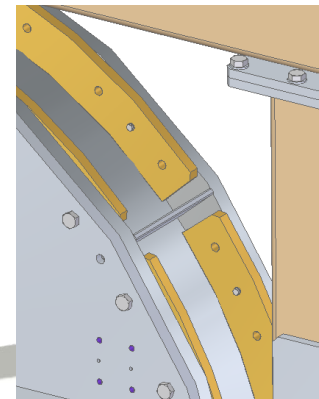
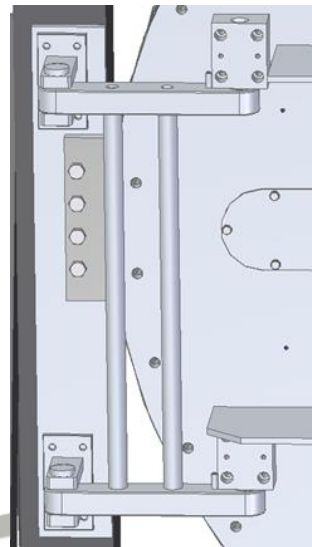
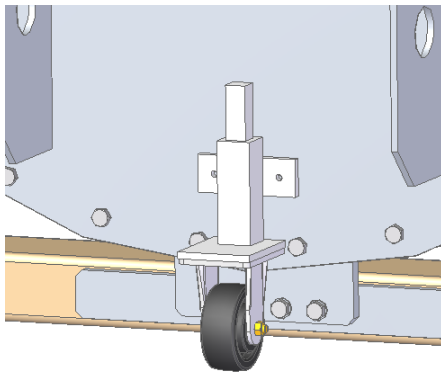
$w(\text{max}) = 1.38 \text{ mm}$   
 $\sigma_{\text{max}} = 35.8 \text{ MPa}$

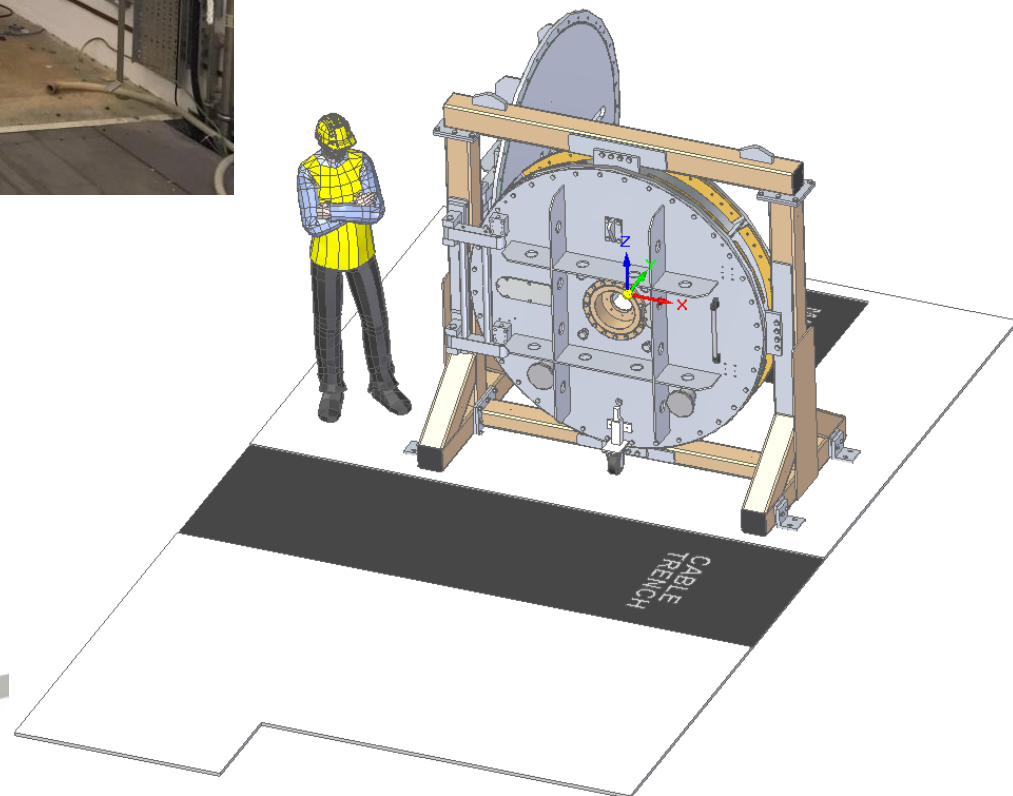
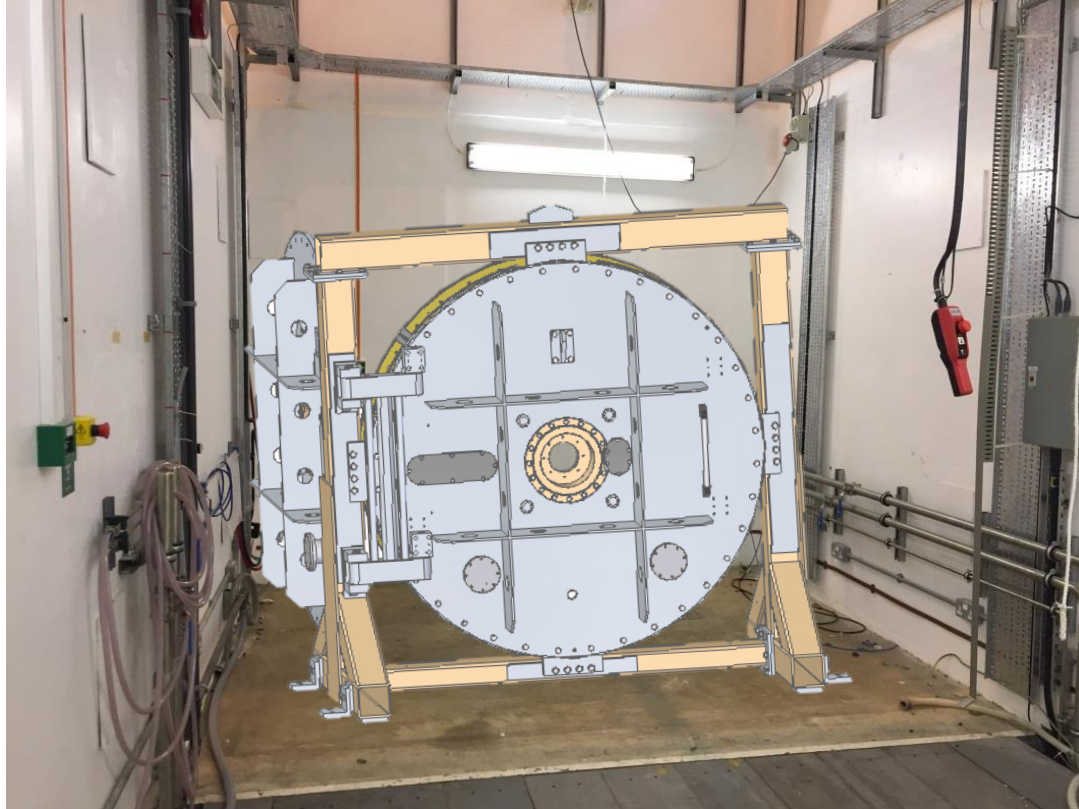
Need reinforcing ribs.



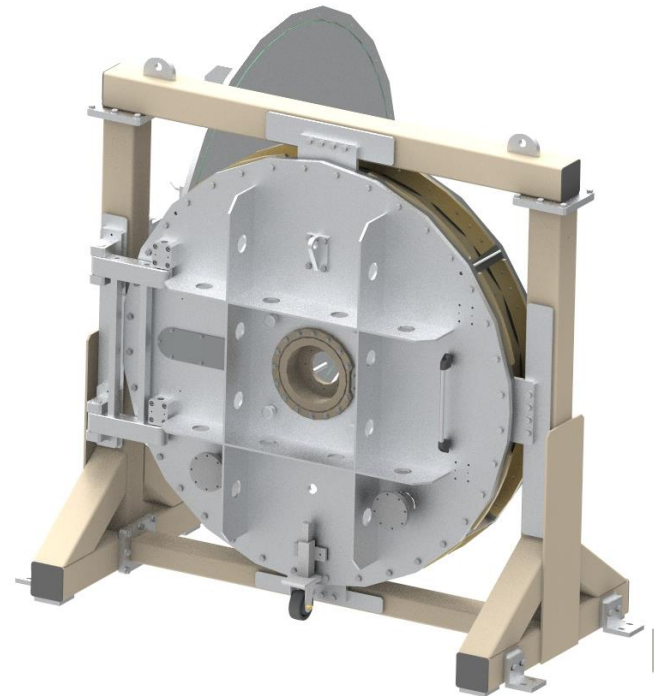
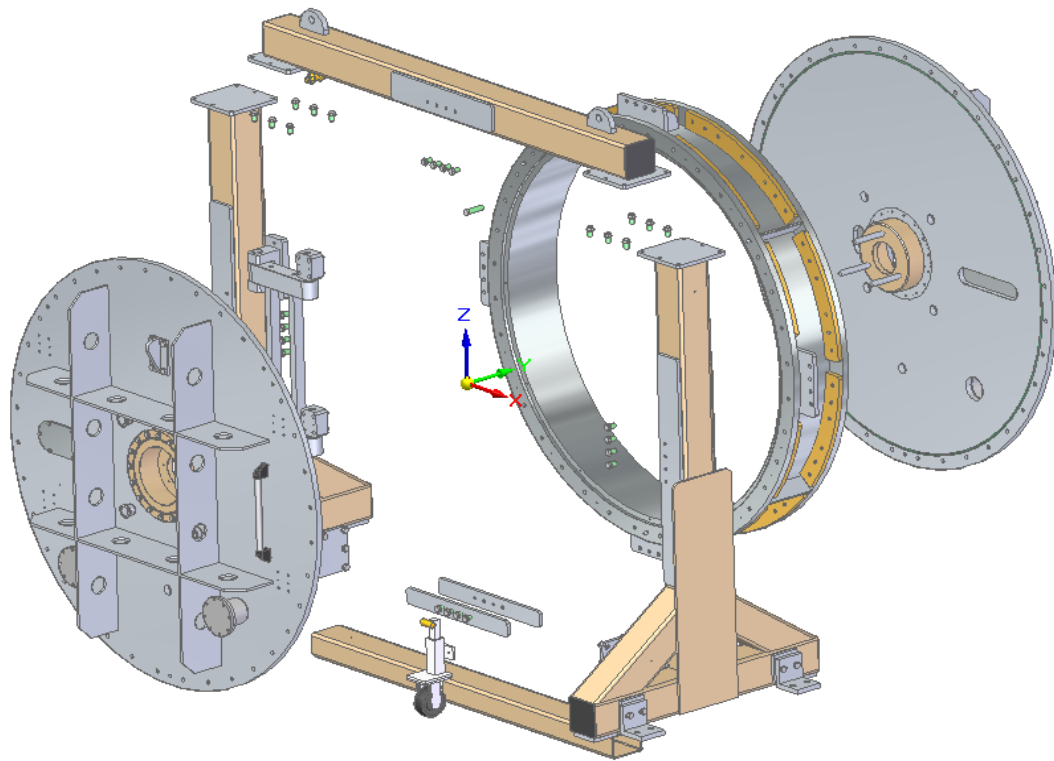
# Further calculations

- Stress in bolts
- Lifting points
- Chamber flanges
- Impact analysis – operations highlighted the importance of safety





# Modular Final Design



# Next Steps

- Drawings have begun and completed those for the frame
- Still need for disc test rig for in-house testing
- Ballistic modelling of failure

## Any Questions?



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